



Aspects of Modeling a Spent Fuel Pool

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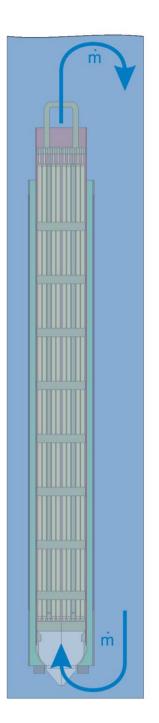


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1. Hydraulic diameter of a fuel assembly

- **2.** Evaporation of a spent fuel pool
- **3.** Transition between different CVs inside the SFP





Convection driven by stack-effect inside fuel assembly (FA)

$$\Delta p_{stack} = g \cdot \rho(T(0)) \cdot (z_{top} - z_{bottom}) - g \int_{z_{bottom}}^{z_{top}} \rho(T(z)) dz$$

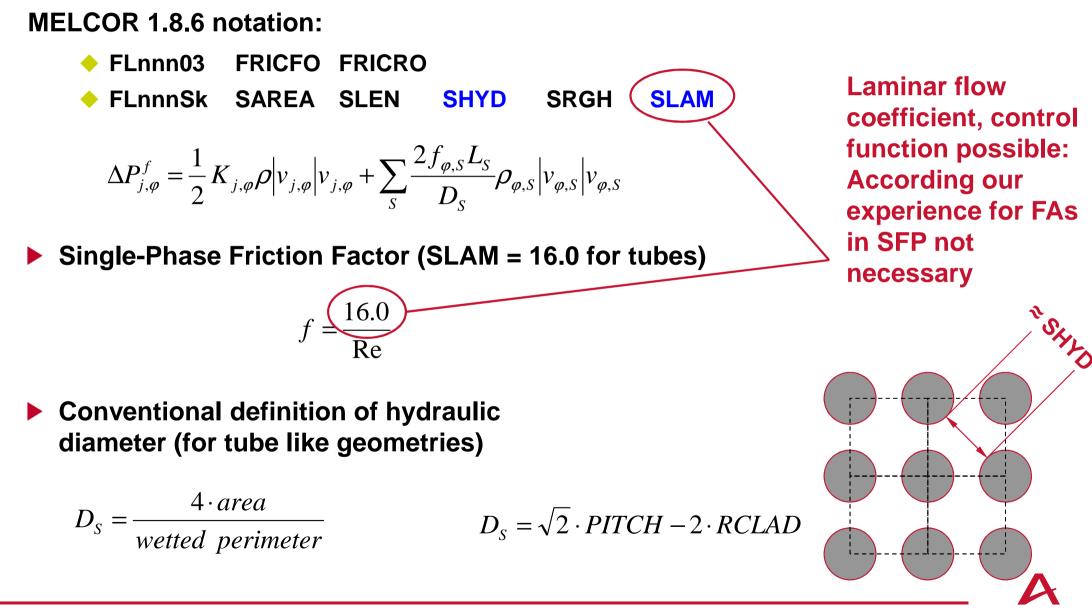
- ► Water flow: Reynolds number ~1000 ➡ more laminar flow
- Air flow: Reynolds number ~1-10 only laminar flow
- Flow limited by wall friction inside fuel assembly

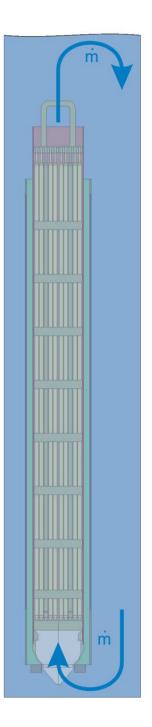
$$\Delta p_{friction} = \frac{64 \cdot \mu}{2 \cdot \rho} \cdot \frac{L_{FA}}{d_{FA}^2 \cdot A_{FA}} \cdot \dot{m}$$

MELCOR

FLnnn03 FRICFO FRICRO is unimportant
FLnnnSk SAREA SLEN SHYD dominates







CFD-Analysis to determine hydraulic diameter (SLAM = 16.0)

Cross section of a complete BWR fuel assembly

Rod bundle

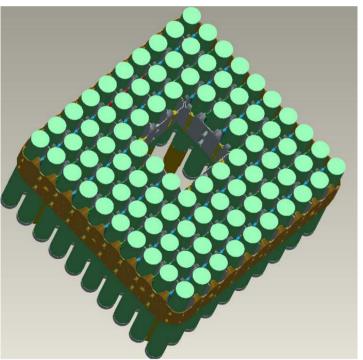
 $\frac{4 \cdot area}{wetted \ perimeter} > \mathbf{SHYD} > \sqrt{2} \cdot PITCH - 2 \cdot RCLAD$

Grid spacer

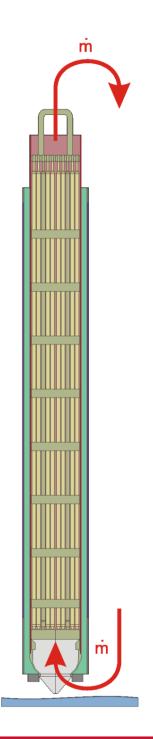
SHYD ≈ 38 % of Bundle-SHYD

Spacer with part of fuel rods (water channel and canister faded-out)

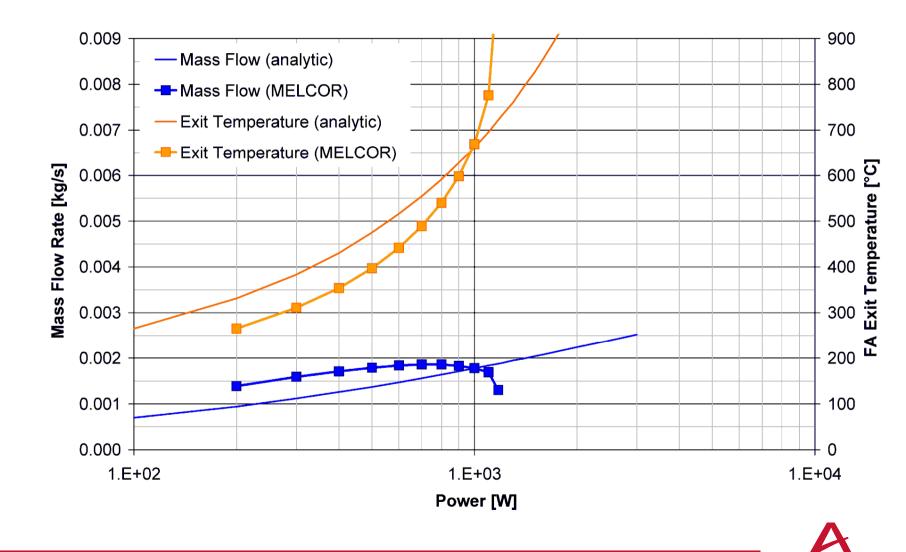
Fuel guard
SHYD ≈ Spacer-SHYD







Air-coolability of FA after dryout of the SFP



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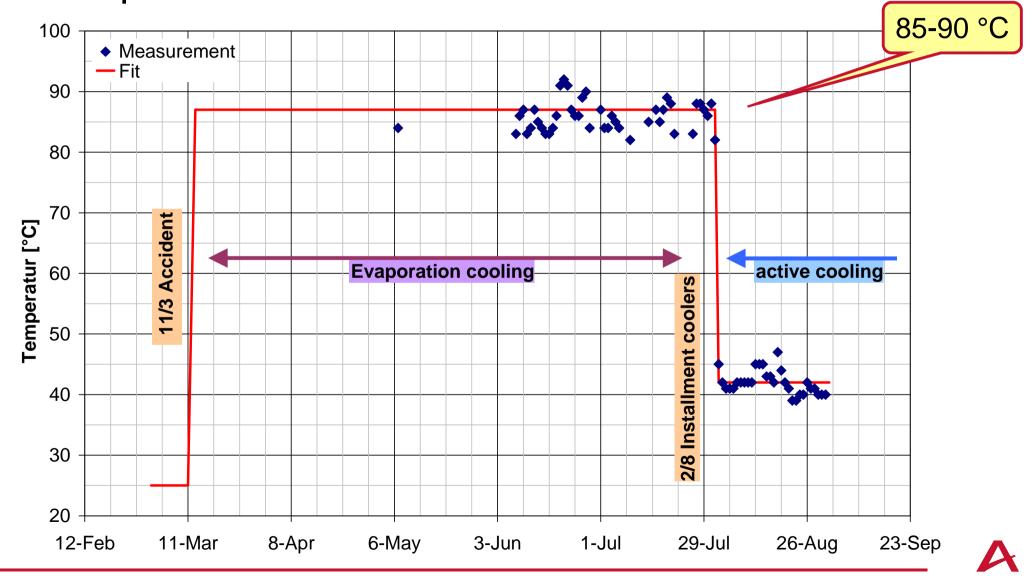




3. Transition between different CVs inside the SFP



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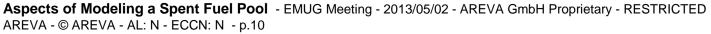


SFP temperature of Fukushima Unit 4

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- Why does Fuel Pool not boil?
 - Low total decay power (1.58 MW @ 6/1/2011)
 - Large water surface (11 m x 12 m)
 - → Evaporation from sub-cooled water surface sufficient to cool pool





AREV

Problem – Weak experimental basis

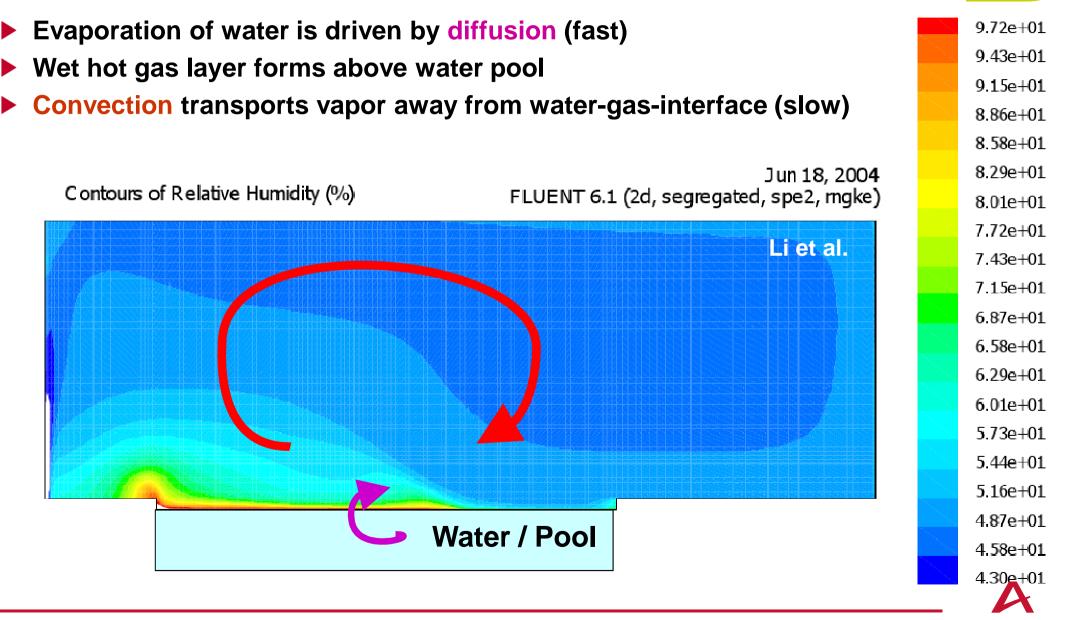
Small scale experiments

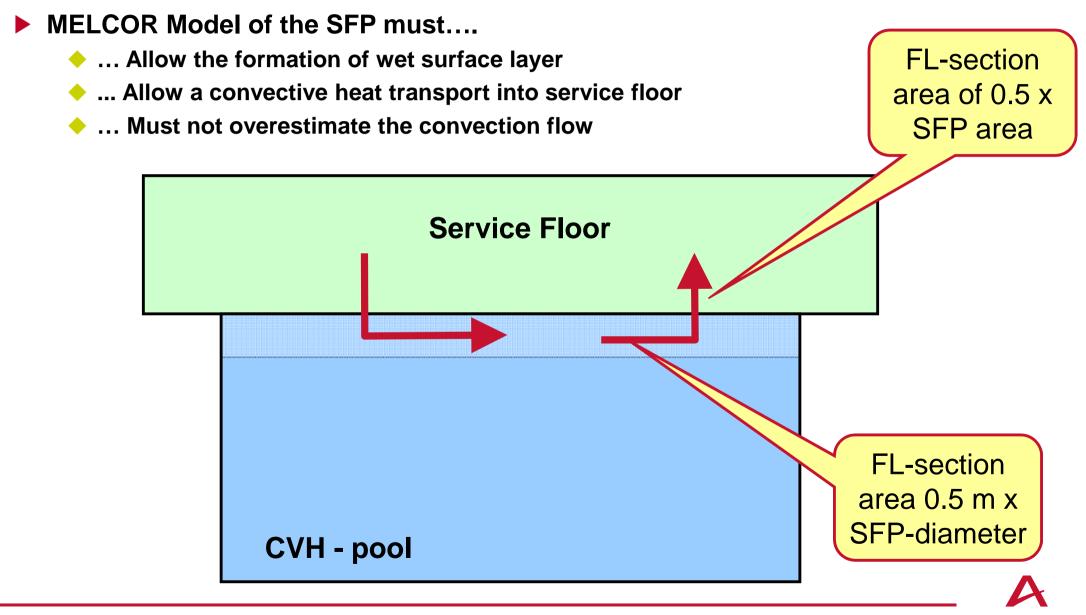
Boelter, Gordon, Griffin Free Evaporation into Air of Water from a Free Horizontal Quiet Surface, Ind. Eng. Chem. 38 (1946) 596–600

Low temperature observations
Z. Li, P. Heiselberg,
CFD Simulations for Water
Evaporation and Airflow Movement
in Swimming Baths, April 2005,
ISSN 1395-7953 R0503)

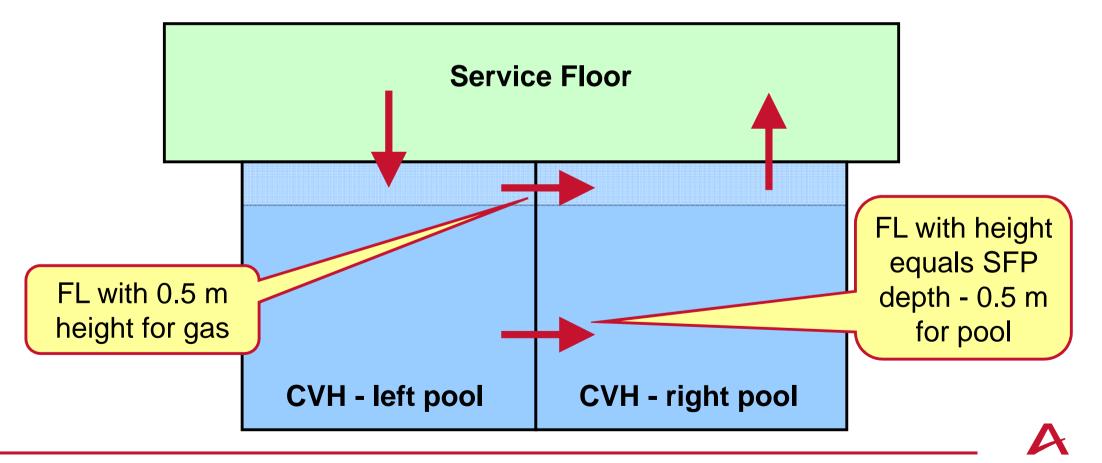






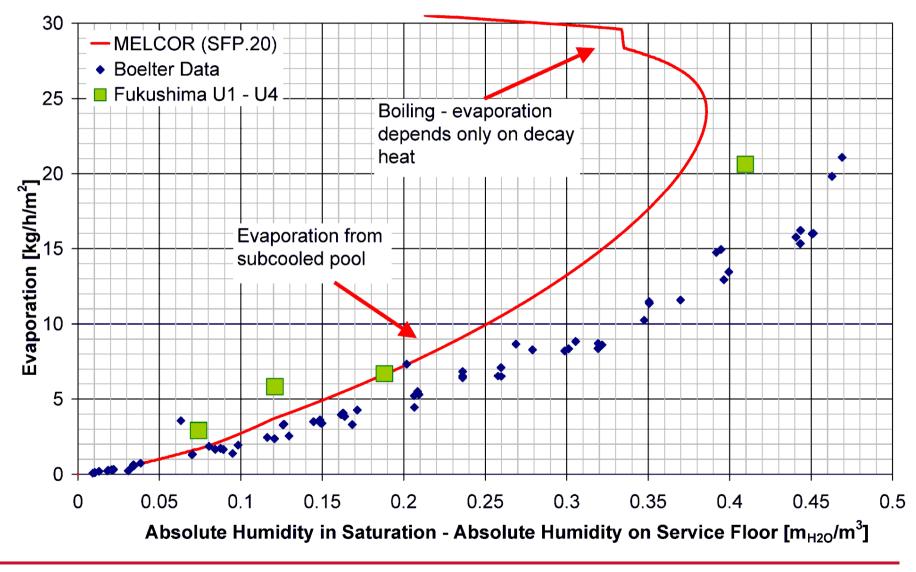


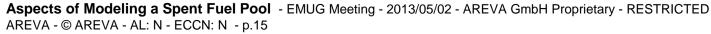
- MELCOR Model of the SFP must....
 - … Allow the formation of wet surface layer
 - In Allow a convective heat transport into service floor
 - Must not overestimate the convection flow



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Resulting evaporation rates in comparison to literature











3. Transition between different CVs inside the SFP



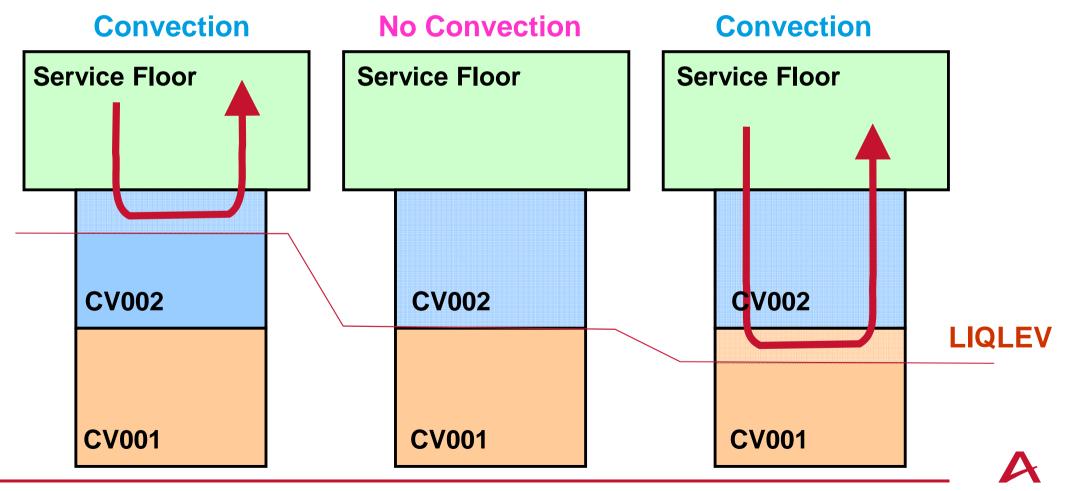
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3.) Transition between different CVs inside the SFP

Problems when SFP is axially separated in different CVH

- No evaporation when LIQLEV equals to a CVH boundary
- Only known fix: Do not axially split SFP



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Conclusions

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Hydraulic diameter of a fuel assembly

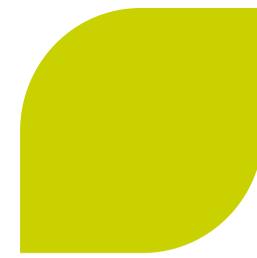
- For small Reynolds numbers hydraulic diameter is very important for accurate pressure losses
- Determination of hydraulic diameter is not straight forward
- Whole range of Reynolds numbers can be satisfactory modeled with appropriate combination of Loss Coefficients and Hydraulic Diameters

Evaporation of a spent fuel pool

- The convection from a layer above the in the spent fuel pool to the service floor limits the evaporation
- 0.5 m layer thickness seams to be reasonable
- Transition between different CVs inside the SFP
 - Horizontal separation of a spent fuel pool is challenging if the level drops
 - We recommend no horizontal sub-division







End of presentation: Aspects of Modeling a Spent Fuel Pool

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